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Prediction of wheel profile wear and rolling contact fatigue for the Stockholm commuter train

Babette Dirks¹, Roger Enblom^{1,2} ¹Royal Institute of Technology, Stockholm - Sweden ²Bombardier Transportation, Västerås - Sweden

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Overview

- Project goals
- Scope of this presentation
- Methods
- Results
- Conclusions



Project

- The main goals of the project are:
 - To create one model for prediction of the total expected life of wheels and rails
 - Selection of reference vehicles, lines and curves for validation of the models
 - Perform/collect measurements
 - Validation of the model
 - Apply the model to investigate the influence of different parameters (wheel/rail profiles, vehicle suspension, axle load, track condition etc.)



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Scope of this presentation

- To study the behavior of two reference vehicles with respect to wear and RCF of the wheels.
 - Two wear and two RCF prediction models have been used in combination with vehicle dynamics simulations.
 - Multi-body simulations in Gensys provided the input to the wear and RCF models



Methods

• Two reference vehicles, running on the Stockholm commuter network, have been selected



Methods – RCF models

- Two RCF initiation prediction models have been studied and compared:
 - **o** 1) based on the shear stress (SI-model)
 - o 2) based on the energy dissipation (DI-model)



Methods – RCF models

• Surface initiated RCF index (SI) of the form:

0

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$$SI = (\tau - k) > 0$$

$$FI = \frac{(\tau - k)}{p} >$$

Shakedown map

τ is the shear stress [N/m²]
k is the yield stress in shear [N/m²]
p is the contact pressure [N/m²]

Methods – RCF models

• Rail RCF model based on T_{γ} (DI-model)



$$T\gamma = T_x \cdot \gamma_x + T_y \cdot \gamma_y$$

 T_x is the longitudinal creep force [N] T_y is the lateral creep force [N] γ_x is the longitudinal creep [-] γ_y is the lateral creep [-]





o 2) Archard's wear model



Pearce and Sheratt (PSH) wear model



Archard wear model (AR)



$$V_w = k \cdot \frac{N \cdot s}{H}$$
 $\Delta z = k \cdot \frac{p_z \cdot \Delta s}{H}$

- V_w = wear volume
- *s* = sliding distance
- N = normal force
- H = hardness
- k = wear coefficient
- Δz = wear depth
- p_z = contact pressure





Results - curving



Methods - simulations

<i>R_m</i> [m]	V _{vehicle} [km/h]	Rail profiles	% of <i>L_{tot}</i>
338	60	1, 2, 3	2.4
432	74	1, 2, 3	2.7
574	92	1, 2, 3	8.0
676	98	1, 2, 3	6.7
895	113	1, 2, 3	6.0
1204	120	1	13.5
2035	120	1	15.5
Straight	120	0	44.9



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Results - RCF

Calculated RCF damage on wheel profile







Results - RCF

 Limitation of the creep forces for high creepages (full slip) for SImodel.



RCF inspections vehicle B

High lateral creep forces







Results - wear

Calculated wear depth on wheel profile







Results - wear

Wear map for single contact in curve



Results - wear

• Calculated wear depth d_z for single contact



Conclusions

- The following main conclusions can be drawn for the RCF prediction models:
 - Both RCF models predict more damage for vehicle B than for vehicle A due to the better steering performance of vehicle A
 - Under poor adhesion conditions, however, the models behave differently:
 - The SI-model predicts less damage for high creepages, due to the independence on creepage
 - Previous research, however, has also shown that high creepage has no effects on RCF life.
- The RCF inspections of the wheels of vehicle B show that the steering of the axles under certain circumstances can be poor.



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Conclusions

- The following main conclusions can be drawn for the wear prediction models:
 - Both wear models predict more wear for vehicle B than for vehicle A due to the better steering performance of vehicle A
 - The Archard's wear model predicts more wear due to the large influence of the sliding velocity in the wear map, therefore, especially for vehicle B.





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Thank you for listening!