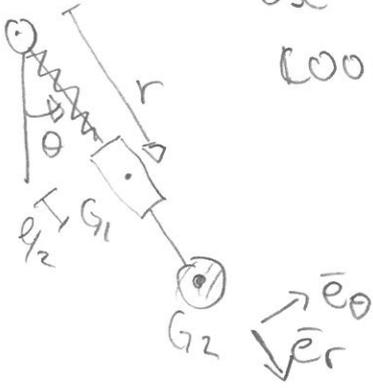


1



Use r, θ as generalized coordinates.

Kinetic energy for the pipe

is $T_1 = \frac{m}{2} |\bar{v}_{G1}|^2 + J_1 \frac{\dot{\theta}^2}{2}$ with

$$\bar{v}_{G1} = \dot{r} \bar{e}_r + r \dot{\theta} \bar{e}_\theta \quad \text{and} \quad J_1 = \frac{m l^2}{12}$$

Kinetic energy for the pipe is $T_2 = \frac{m}{2} |\bar{v}_{G2}|^2$
with $\bar{v}_{G2} = 3l \dot{\theta} \bar{e}_\theta$.

$$T = T_1 + T_2 = \frac{m}{2} \left(\dot{r}^2 + \left(r^2 + \frac{109}{12} l^2 \right) \dot{\theta}^2 \right)$$

Potential energy: $V_1^g = -m g r \cos \theta$ $V_2^g = -m g 3l \cos \theta$

$$V^k = \frac{k}{2} \left(\underbrace{r - \frac{l}{2}}_{\text{pipe end}} - \underbrace{l}_{\text{natural length}} \right)^2 = \frac{m g}{l} \left(r - \frac{3l}{2} \right)^2$$

$$V = V_1^g + V_2^g + V^k = \frac{m g}{l} \left[-l(r+3l) \cos \theta + \left(r - \frac{3l}{2} \right)^2 \right]$$

Equilibrium points r_0, θ_0 satisfies

$$\frac{\partial V}{\partial r} \Big|_{r_0, \theta_0} = \frac{m g}{l} \left[-l \cos \theta_0 + 2 \left(r_0 - \frac{3l}{2} \right) \right] = 0$$

$$\frac{\partial V}{\partial \theta} \Big|_{r_0, \theta_0} = \frac{m g}{l} l (r_0 + 3l) \sin \theta_0 = 0 \quad \text{We find } r_0 = 2l, \theta_0 = 0$$

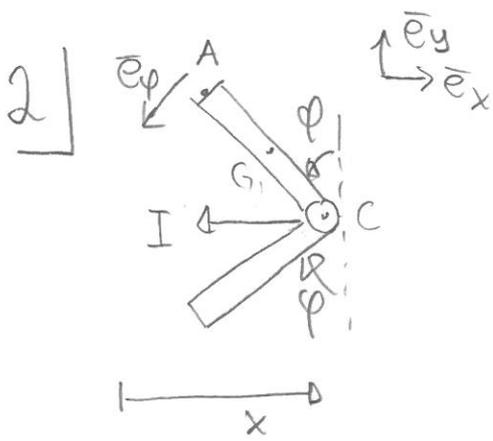
or $(r_0 = l, \theta_0 = \pi)$. Only the first is likely to be stable.

The mass matrix is $M = \frac{\partial^2 T}{\partial (\dot{r}, \dot{\theta})^2} \Big|_{r_0, \theta_0} = m \begin{bmatrix} 1 & 0 \\ 0 & \frac{157}{12} l^2 \end{bmatrix}$

The stiffness matrix is $K = \frac{\partial^2 V}{\partial (r, \theta)^2} \Big|_{r_0, \theta_0} = \frac{m g}{l} \begin{bmatrix} 2 & 0 \\ 0 & 5l^2 \end{bmatrix}$

Since the matrices are diagonal, we find

$$\boxed{\omega_1^2 = \frac{2g}{l}} \quad (\text{radial osc}) \quad \text{and} \quad \boxed{\omega_2^2 = \frac{60g}{157l}} \quad (\text{angular osc})$$



By symmetry, the point C moves only in the x direction and both rods make the same angle φ .

Use x, φ as generalized coordinates. For the rod AC:

$$T_1 = \frac{m}{2} |\bar{v}_{G_1}|^2 + J_1 \dot{\varphi}^2 \quad \text{with}$$

$$\bar{v}_{G_1} = \dot{x} \bar{e}_x + \frac{l}{2} \dot{\varphi} \bar{e}_\varphi \quad \text{and} \quad J_1 = \frac{ml^2}{12}$$

Using $\bar{e}_x \cdot \bar{e}_\varphi = -\cos\varphi$ we find $T_1 = \frac{m}{2} (\dot{x}^2 - l \cos\varphi \dot{x} \dot{\varphi} + \frac{l^2}{4} \dot{\varphi}^2) + \frac{ml^2}{12} \frac{\dot{\varphi}^2}{2}$

$$T = T_1 + T_2 = 2T_1 = m \left[\dot{x}^2 - l \cos\varphi \dot{x} \dot{\varphi} + \frac{l^2}{3} \dot{\varphi}^2 \right]$$

Since $\bar{v}_C = \bar{e}_x \dot{x} + \bar{0} \dot{\varphi}$ we find the generalized impulses $I_x = \bar{e}_x \cdot (-I \bar{e}_x) = -I$, $I_\varphi = \bar{0} \cdot (-I \bar{e}_x) = 0$

Lagrange's equations for impact are

$$\frac{\partial T}{\partial \dot{x}} \Big|_{\dot{x}_f, \dot{\varphi}_f} - \frac{\partial T}{\partial \dot{x}} \Big|_{\dot{x}_i, \dot{\varphi}_i} = I_x \quad \frac{\partial T}{\partial \dot{\varphi}} \Big|_{\dot{x}_f, \dot{\varphi}_f} - \frac{\partial T}{\partial \dot{\varphi}} \Big|_{\dot{x}_i, \dot{\varphi}_i} \quad \text{with} \quad \dot{x}_i = v_0, \dot{\varphi}_i = 0$$

The second equation is

$$m \left[-l \cos\varphi_0 \dot{x}_f + \frac{2l^2}{3} \dot{\varphi}_f \right] - m \left[-l \cos\varphi_0 v_0 + \frac{2l^2}{3} \cdot 0 \right] = 0$$

$$\Rightarrow \dot{\varphi}_f = \frac{3}{2} \frac{\cos\varphi_0}{l} (\dot{x}_f - v_0) \quad (*) \quad \text{Elastic impact means } \dot{x}_f = -v_0.$$

$$\Rightarrow \dot{\varphi}_f = -\frac{3 \cos\varphi_0 v_0}{l}. \quad \text{This gives } \boxed{\bar{v}_{A_f} = \dot{x}_f \bar{e}_x + l \dot{\varphi}_f \bar{e}_\varphi =}$$

$$= -v_0 \bar{e}_x - 3 \cos\varphi_0 v_0 \bar{e}_\varphi = v_0 (-\bar{e}_x - 3 \cos\varphi_0 \bar{e}_\varphi) =$$

$$= v_0 \left[(-1 + 3 \cos^2\varphi_0) \bar{e}_x + 3 \sin\varphi_0 \cos\varphi_0 \bar{e}_y \right].$$

Instead of using $\dot{x}_f = -v_0$ we could have used

$$T_f - T_i = \{ (*) \} = m \left[1 - \frac{3}{4} \cos^2\varphi_0 \right] (\dot{x}_f^2 - v_0^2) = 0 \quad \text{to conclude}$$

that $\dot{x}_f = -v_0$.

3



Use cylindrical coordinates:

$$\bar{v}_G = (R-a)\dot{\theta}\bar{e}_\theta + \dot{z}\bar{e}_z$$

$$\bar{\omega} = \omega_r\bar{e}_r + \omega_\theta\bar{e}_\theta + \omega_z\bar{e}_z$$

$$\bar{p} = m\bar{v}_G = m[(R-a)\dot{\theta}\bar{e}_\theta + \dot{z}\bar{e}_z]$$

$$\bar{L}_G = \{\text{spherical symmetry}\} = J_G\bar{\omega} = \frac{2m}{5}a^2\bar{\omega}$$

Call the contact force $\bar{R} = -N\bar{e}_r + \bar{F}_\mu$.

Momentum balance:

$$\dot{\bar{p}} = -mg\bar{e}_z + \bar{R} \quad (1)$$

Angular momentum balance:

$$\dot{\bar{L}}_G = \bar{r}_{Gc} \times \bar{R} = a\bar{e}_r \times \bar{R} \quad (2)$$

Eliminate \bar{R} :

$$(2) - \bar{r}_{Gc} \times (1) \Rightarrow \dot{\bar{L}}_G - a\bar{e}_r \times \dot{\bar{p}} = a\bar{e}_r \times mg\bar{e}_z = -mga\bar{e}_\theta$$

Either use the known $\dot{\bar{e}}_r = \dot{\theta}\bar{e}_\theta$, $\dot{\bar{e}}_\theta = -\dot{\theta}\bar{e}_r$ or

use $\frac{d\bar{A}}{dt} = \left(\frac{d\bar{A}}{dt}\right)_{r,\varphi,z} + \bar{\omega}_{r,\varphi,z} \times \bar{A}$ with $\bar{\omega}_{r,\varphi,z} = \dot{\theta}\bar{e}_z$

to get

$$\dot{\bar{L}}_G - a\bar{e}_r \times \dot{\bar{p}} = \frac{2ma^2}{5}(\dot{\omega}_r\bar{e}_r + \dot{\omega}_\theta\bar{e}_\theta + \dot{\omega}_z\bar{e}_z + \omega_r\dot{\theta}\bar{e}_\theta - \omega_\theta\dot{\theta}\bar{e}_r) -$$

$$- a\bar{e}_r \times m[(R-a)\ddot{\theta}\bar{e}_\theta + \ddot{z}\bar{e}_z - (R-a)\dot{\theta}^2\bar{e}_r] = -mga\bar{e}_\theta$$

We find

$$\bar{e}_r: \frac{2ma^2}{5}(\dot{\omega}_r - \dot{\theta}\omega_\theta) = 0 \quad (3)$$

$$\bar{e}_\theta: \frac{2ma^2}{5}(\dot{\omega}_\theta + \dot{\theta}\omega_r) + ma\ddot{z} = -mga \quad (4)$$

$$\bar{e}_z: \frac{2ma^2}{5}\dot{\omega}_z - ma(R-a)\ddot{\theta} = 0 \quad (5)$$

3] cont.

Then we have the rolling condition:

$$\vec{v}_G + \vec{\omega} \times \vec{r}_{GC} = \vec{0} \Rightarrow (R-a)\dot{\theta}\vec{e}_\theta + \dot{z}\vec{e}_z + (\omega_r\vec{e}_r + \omega_\theta\vec{e}_\theta + \omega_z\vec{e}_z) \times a\vec{e}_r = \vec{0}$$

$$\vec{e}_r: 0 = 0$$

$$\vec{e}_\theta: (R-a)\dot{\theta} + a\omega_z = 0 \quad (6)$$

$$\vec{e}_z: \dot{z} - a\omega_\theta = 0 \quad (7)$$

(3)-(7) determine the motion.

$$\frac{d}{dt} (6) \text{ in } (5) \text{ gives } \ddot{\theta} = 0 \text{ so } \boxed{\dot{\theta}(t) = \dot{\theta}(0) = \underline{\Omega}}$$

$$\boxed{\omega_z(t) = -\frac{R-a}{a}\Omega} \text{ for some constant } \underline{\Omega}.$$

Assuming constant z , (7) $\Rightarrow \omega_\theta = 0$

Using this in (3), (4) gives

$$\dot{\omega}_r = 0, \quad \frac{2mg^2}{5}\Omega\omega_r = -mga \text{ so if } \Omega \neq 0 \text{ we}$$

get a constant z solution

$$\boxed{\omega_r(t) = -\frac{5g}{2a\Omega}, \quad \omega_\theta(t) = 0, \quad z(t) = z_0}$$

3 cont) General solution:

If $\Omega = 0$ we find $\dot{\theta} = 0, \dot{\omega}_r = 0, \ddot{z} = -\frac{5}{7}g$ but this means rolling without normal force, which seems rather unphysical, and any disturbance will turn it into free fall $\ddot{z} = -g$.

If $\Omega \neq 0$ we use $\frac{d}{dt}(7)$ to eliminate \ddot{z} from (4) and $\frac{d}{dt}(3)$ to eliminate $\dot{\omega}_\theta$ from (4) to get

$$\ddot{\omega}_r + \frac{2}{7}\Omega^2 \omega_r = -\frac{5g\Omega}{7a} \Rightarrow \left\{ \omega_n = \sqrt{\frac{2}{7}}\Omega \right\}$$

$$\omega_r(t) = A \cos(\omega_n t) + B \sin(\omega_n t) - \frac{5g}{2a\Omega}$$

$$z(t) = \int (7), (3) \Rightarrow \dot{z} = \frac{a\dot{\omega}_r}{\Omega} \Rightarrow z = \frac{a}{\Omega} \omega_r(t) + C$$

Thus z will oscillate vertically, while θ changes linearly in time.

4]

The point A is also the center of mass. The x, y, z axes are principal axes for $\bar{\bar{J}}_A$ and they are equivalent, so $\bar{\bar{J}}_A$ is spherical and

$$\bar{\bar{J}}_A = J_A \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad \text{with } J_A = J_{Axx} = 0 + \frac{ml^2}{12} + \frac{ml^2}{12} = \frac{ml^2}{6}$$

Moving to the point O, we get

$$\bar{\bar{J}}_O = \bar{\bar{J}}_A + 3m \begin{bmatrix} y_A^2 + z_A^2 & -x_A y_A & -x_A z_A \\ -y_A x_A & x_A^2 + z_A^2 & -y_A z_A \\ -z_A x_A & -z_A y_A & y_A^2 + z_A^2 \end{bmatrix} =$$

$$= \frac{ml^2}{6} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} + \frac{3ml^2}{4} \begin{bmatrix} 2 & -1 & -1 \\ -1 & 2 & -1 \\ -1 & -1 & 2 \end{bmatrix} =$$

$$= ml^2 \begin{bmatrix} 5/3 & -3/4 & -3/4 \\ -3/4 & 5/3 & -3/4 \\ -3/4 & -3/4 & 5/3 \end{bmatrix}$$

Since $\bar{\bar{J}}_A$ is spherical, the $(\bar{e}_x + \bar{e}_y + \bar{e}_z)$ axis is (inertially) an axis of rotation symmetry,

and we find $\bar{\bar{J}}_O \begin{bmatrix} 1 \\ 1 \\ 1 \end{bmatrix} = \frac{ml^2}{6} \begin{bmatrix} 1 \\ 1 \\ 1 \end{bmatrix}$ so $J_1 = \frac{ml^2}{6}$

We also know from rotation symmetry that

$J_2 = J_3$ for any two axes perpendicular to $\bar{e}_x + \bar{e}_y + \bar{e}_z$,

for example $\bar{e}_x - \bar{e}_z$ and $\bar{e}_x - 2\bar{e}_y + \bar{e}_z$.

Also $J_1 + J_2 + J_3 = \text{Trace}(\bar{\bar{J}}_O) = 5ml^2$

$$J_2 = J_3 = \frac{29}{12} ml^2$$

$$5] \quad T = \frac{m}{2} (\dot{r}^2 + r^2 \dot{\theta}^2) \quad V = \frac{k}{2} (r-l)^2$$

$$L = \frac{m}{2} (\dot{r}^2 + r^2 \dot{\theta}^2) - \frac{k}{2} (r-l)^2$$

$$\frac{\partial L}{\partial \theta} = 0 \Rightarrow \theta \text{ is cyclic} \Rightarrow p_{\theta} = \frac{\partial L}{\partial \dot{\theta}} = m r^2 \dot{\theta} \quad (1)$$

The r Lagrange's equation is

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{r}} \right) - \frac{\partial L}{\partial r} = m(\ddot{r} - r\dot{\theta}^2) + k(r-l) = 0 \quad (2)$$

Assuming a relative eq point $r(t) = r_0$ $\dot{\theta}(t) = \Omega_0$

we find $p_{\theta} = m r_0^2 \Omega_0$ and $-m r_0 \Omega_0^2 + k(r_0 - l) = 0$

$$\Rightarrow \boxed{\Omega_0^2 = \frac{k}{m} \frac{r_0 - l}{r_0}, \quad r_0 \geq l} \text{ This also gives}$$

$$\boxed{(p_{\theta})_0^2 = m^2 r_0^4 \Omega_0^2 = m^2 r_0^4 \frac{k}{m} \frac{r_0 - l}{r_0} = \underline{km r_0^3 (r_0 - l)}} \text{ .}$$

We can now use (1) to eliminate $\dot{\theta}^2$ from (2):

$$(1) \Rightarrow \dot{\theta}^2 = \frac{(p_{\theta})_0^2}{m^2 r^4} = \frac{k}{m} \frac{r_0^3}{r^4} (r_0 - l) \text{ . Insert into (2):}$$

$$m(\ddot{r} - \frac{k}{m} \frac{r_0^3}{r^3} (r_0 - l)) + k(r-l) = 0 \Rightarrow$$

$$\ddot{r} + \frac{k}{m} \underbrace{\left[(r-l) - \frac{r_0^3}{r^3} (r_0 - l) \right]}_{f(r)} = 0 \text{ . Clearly } f(r_0) = 0 \text{ and}$$

$$f'(r_0) = 1 + 3 \frac{(r_0 - l)}{r_0} \text{ . The linearized equation is}$$

$$\ddot{r} + \underbrace{\frac{k}{m} \left(1 + 3 \frac{(r_0 - l)}{r_0} \right)}_{\omega^2} (r - r_0) = 0 (r - r_0)^2 \text{ and}$$

we find the oscillation frequency $\boxed{\omega^2 = \frac{k}{m} \left(1 + 3 \frac{(r_0 - l)}{r_0} \right)}$

or in terms of the rotation speed Ω_0

$$\omega^2 = \frac{k}{m} + 3 \Omega_0^2 \text{ . [Compare with problem 23 in the problem collection, but where rotation speed is fixed.]}$$

6]

$$\delta \int_{t_0}^{t_1} \left[\sum_a \dot{q}_a p_a - H(q, p, t) \right] dt = \left\{ t_0, t_1 \text{ fixed} \right\} =$$

$$= \int_{t_0}^{t_1} \sum_a \left(\delta \dot{q}_a p_a + \dot{q}_a \delta p_a - \frac{\partial H}{\partial q_a} \delta q_a - \frac{\partial H}{\partial p_a} \delta p_a \right) dt =$$

$$= \left\{ \text{partial integration on } \delta \dot{q}_a p_a \right\} =$$

$$= \left[\sum_a \delta q_a p_a \right]_{t_0}^{t_1} + \int_{t_0}^{t_1} \sum_a \left(-\delta q_a \dot{p}_a + \dot{q}_a \delta p_a - \frac{\partial H}{\partial q_a} \delta q_a - \frac{\partial H}{\partial p_a} \delta p_a \right) dt$$

$$= \left\{ q_a(t_1), q_a(t_0) \text{ fixed} \right\} =$$

$$= \int_{t_0}^{t_1} \sum_a \left[\delta q_a \left(-\dot{p}_a - \frac{\partial H}{\partial q_a} \right) + \delta p_a \left(\dot{q}_a - \frac{\partial H}{\partial p_a} \right) \right] dt$$

By the varying each component of δq_a , δp_a independently, and using the standard argument (see course text) we find $-\dot{p}_a - \frac{\partial H}{\partial q_a} = 0$ $\dot{q}_a - \frac{\partial H}{\partial p_a} = 0$ for $t_0 \leq t \leq t_1$.