



Boundary Conditions

Initial conditions

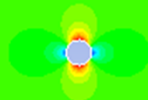


- Steady-state problem -> completely determined by BC
- Limit-cycle oscillation -> completely determined by BC
- Transient computation of an initial-value problem is dependent on the initial conditions
- Initial conditions may be important for the convergence to steady state
 - Set to constant value
 - Solve simplified equation (potential flow – “hybrid”)
 - Solve on coarser grids (FMG = full multi-grid)

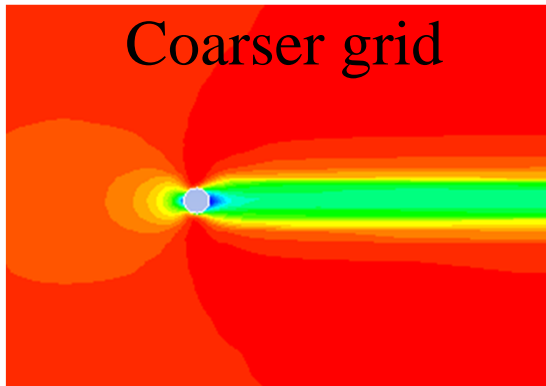
Constant value



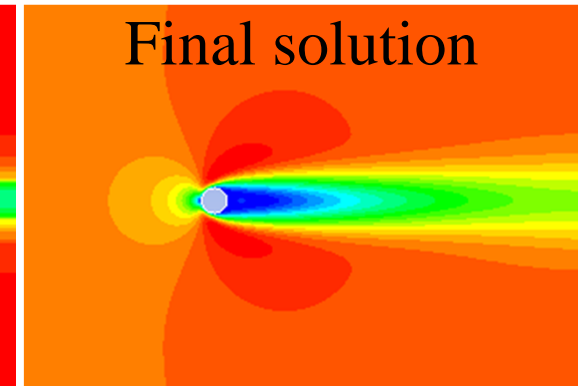
Potential flow



Coarser grid



Final solution



Boundary flow field



- Supersonic inflow
 - All information from boundary to interior
- Subsonic inflow
 - Most information from boundary to interior
 - pressure related information from interior to boundary.
- Subsonic outflow
 - Only pressure related information from boundary to interior
- Supersonic outflow
 - No information from boundary to interior
- Must be
 - Numerically stable
 - Well posed

Boundary influence

If interaction between the boundary and interior flow fields is a problem:

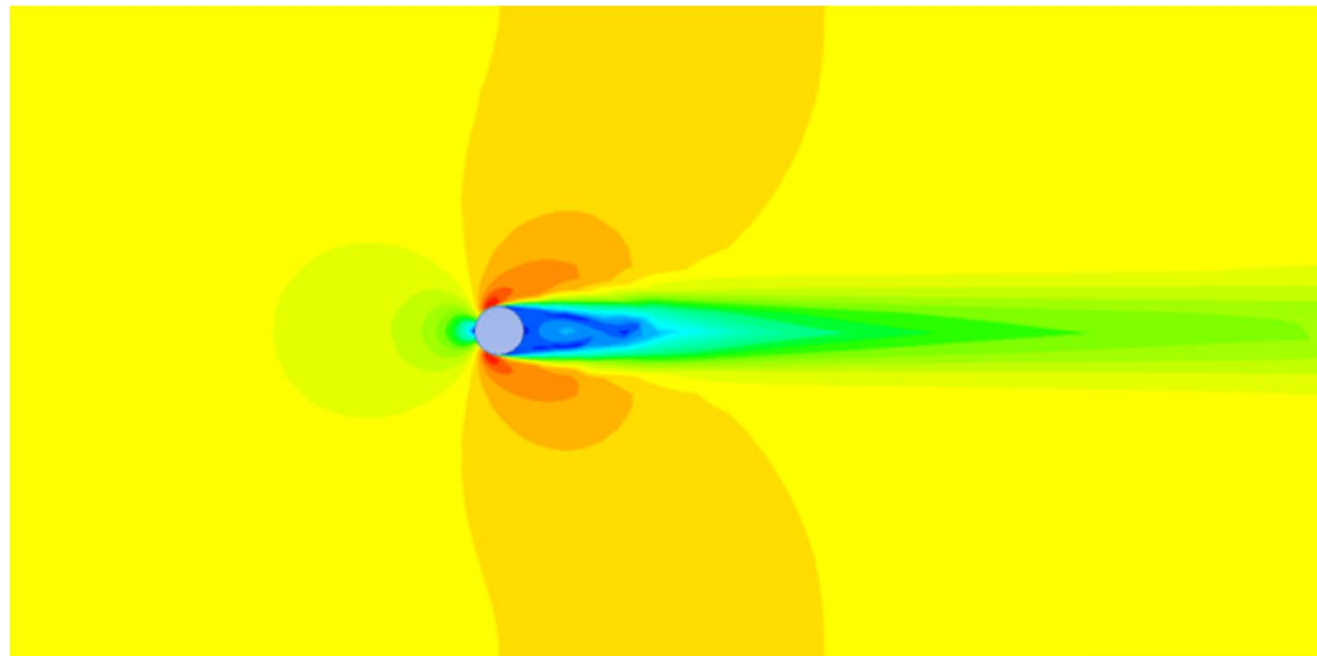
- Extend the boundaries for external flows
 - typically 10-50 times the size of the object
 - Less problem in 3D flows
- Include more of the true inflow/outflow geometries for internal flows
- Make empirical/mathematical corrections on the boundaries
- Warning for recirculation
 - Inflow at outflow boundaries, or
 - Outflow at inflow boundaries
- **Be aware of the problem!!!**



External flow B.C.

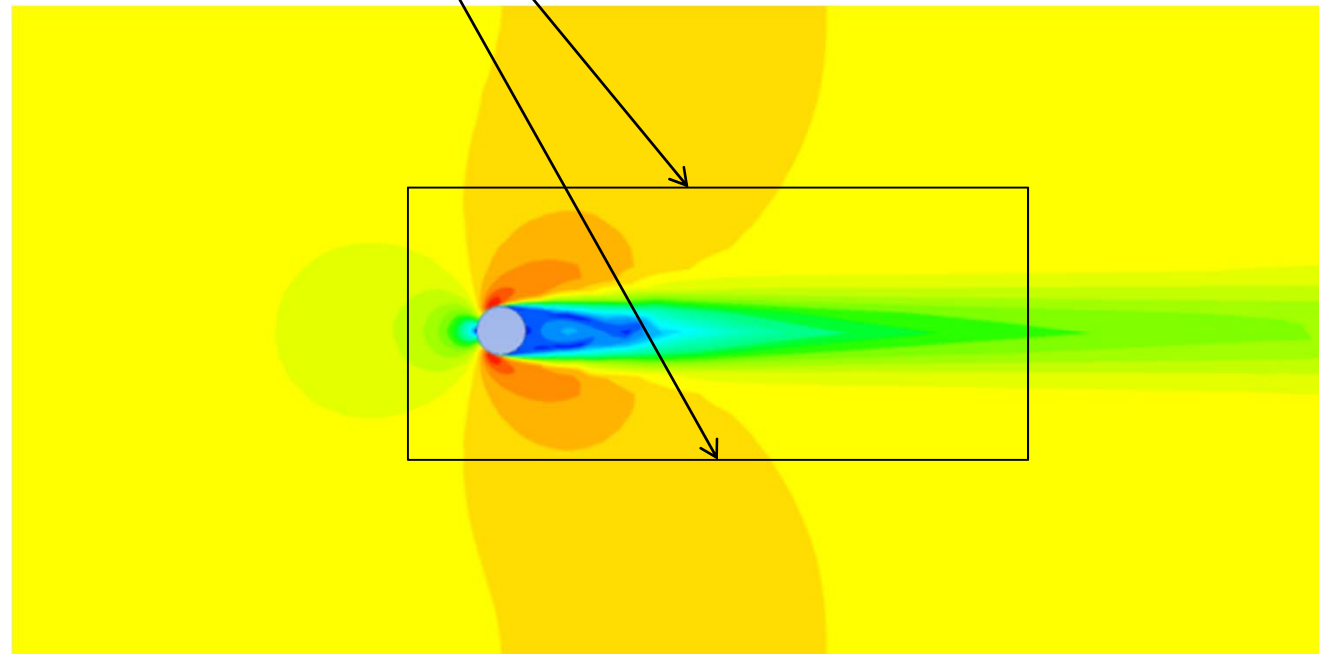
Example: Cylinder wake (picture, thanks to one of you)

- Real problem: one isolated cylinder in free air
- How to define the domain and boundary conditions?



External flow B.C.

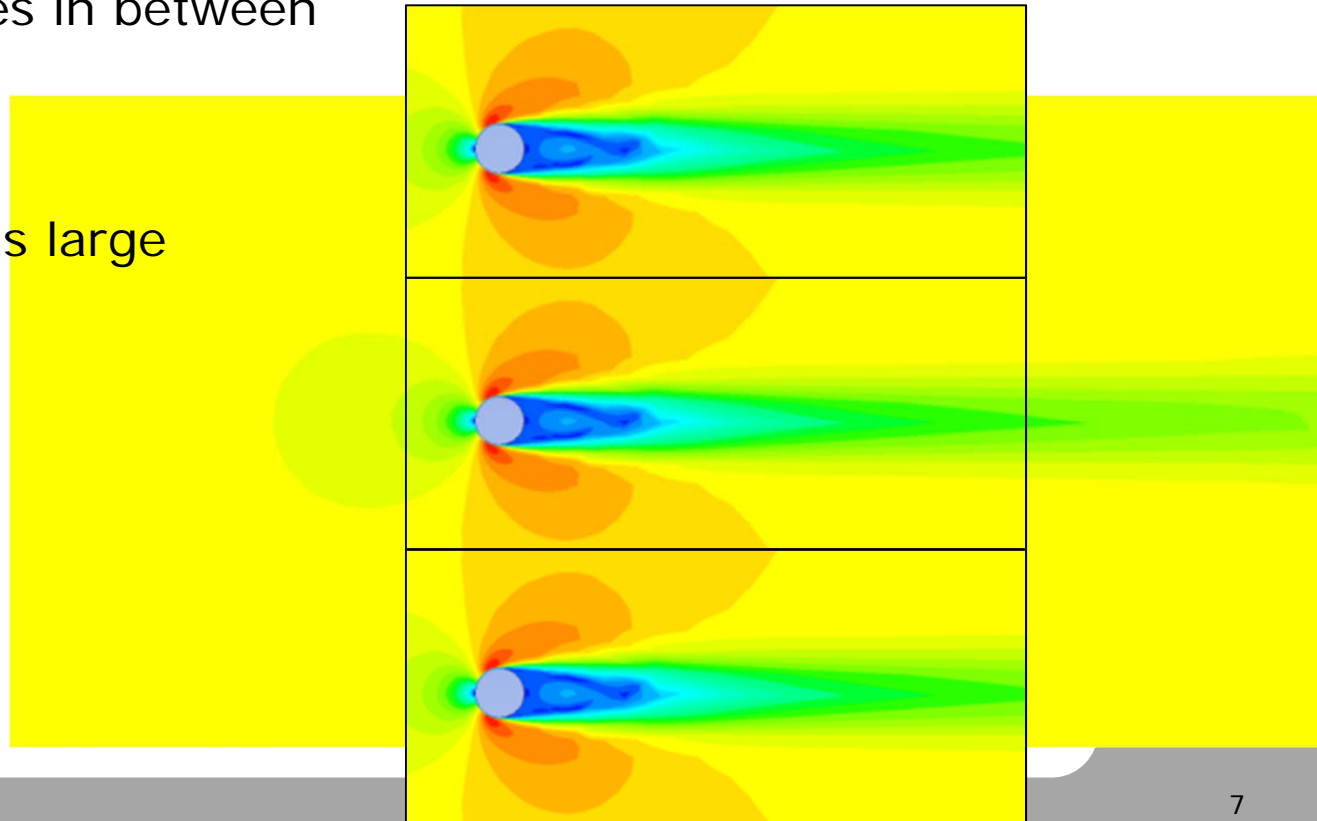
As symmetry or periodic BCs



External flow B.C.

As symmetry or periodic BCs

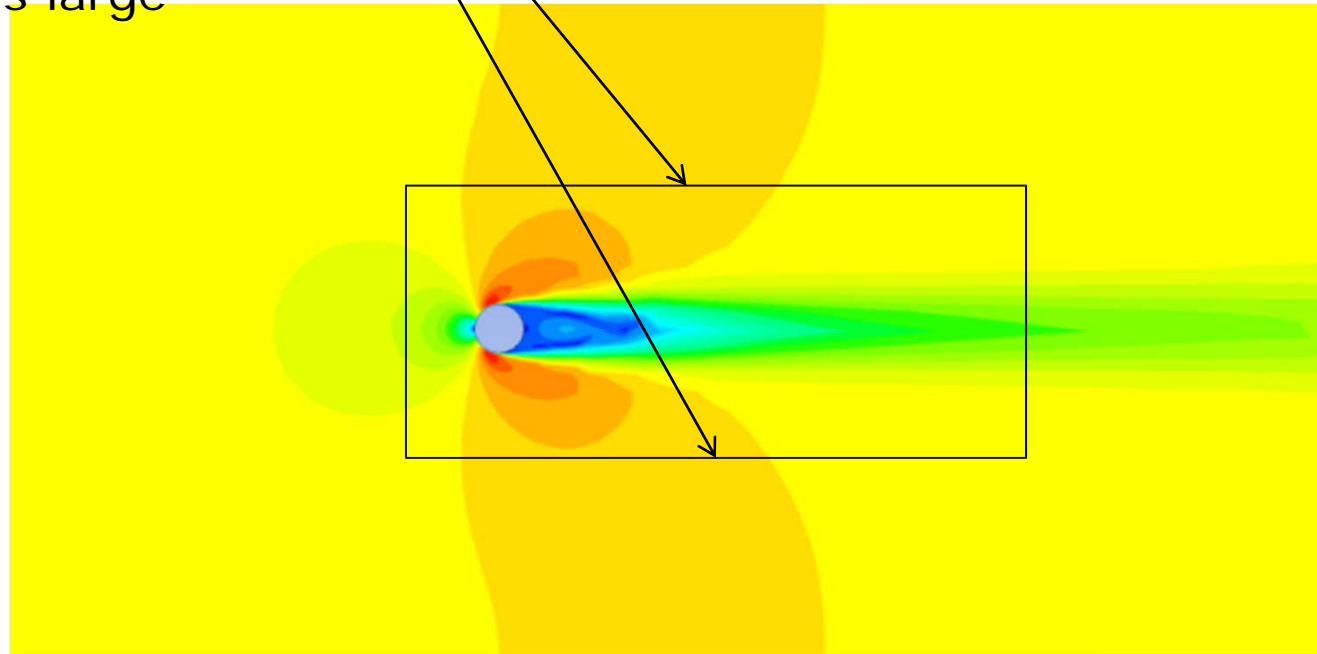
- Corresponds to a row of cylinders
- Distance set by domain size
- Flow accelerates in between
- Higher forces
- OK if distance is large



External flow B.C.

As pressure BCs (pressure outlet)

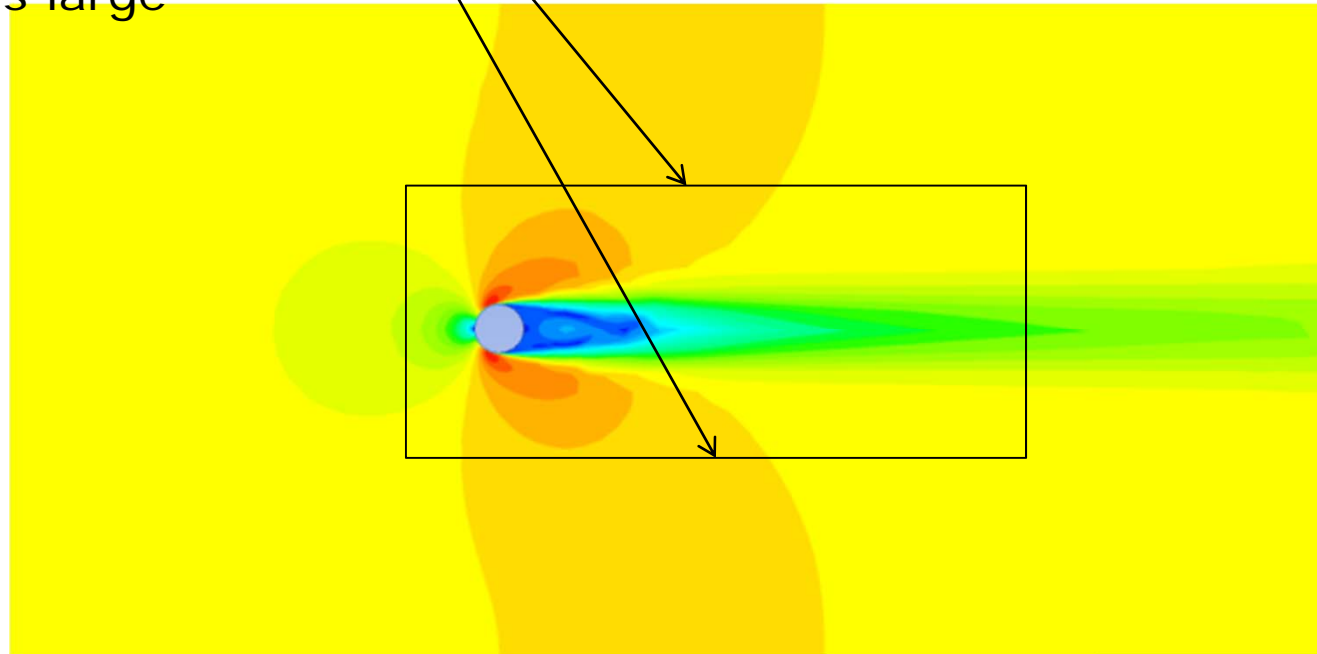
- Velocity and pressure not constant at the boundary
- Might be a local inflow – numerical instability?
- OK if distance is large



External flow B.C.

As inlet BCs (velocity inlet)

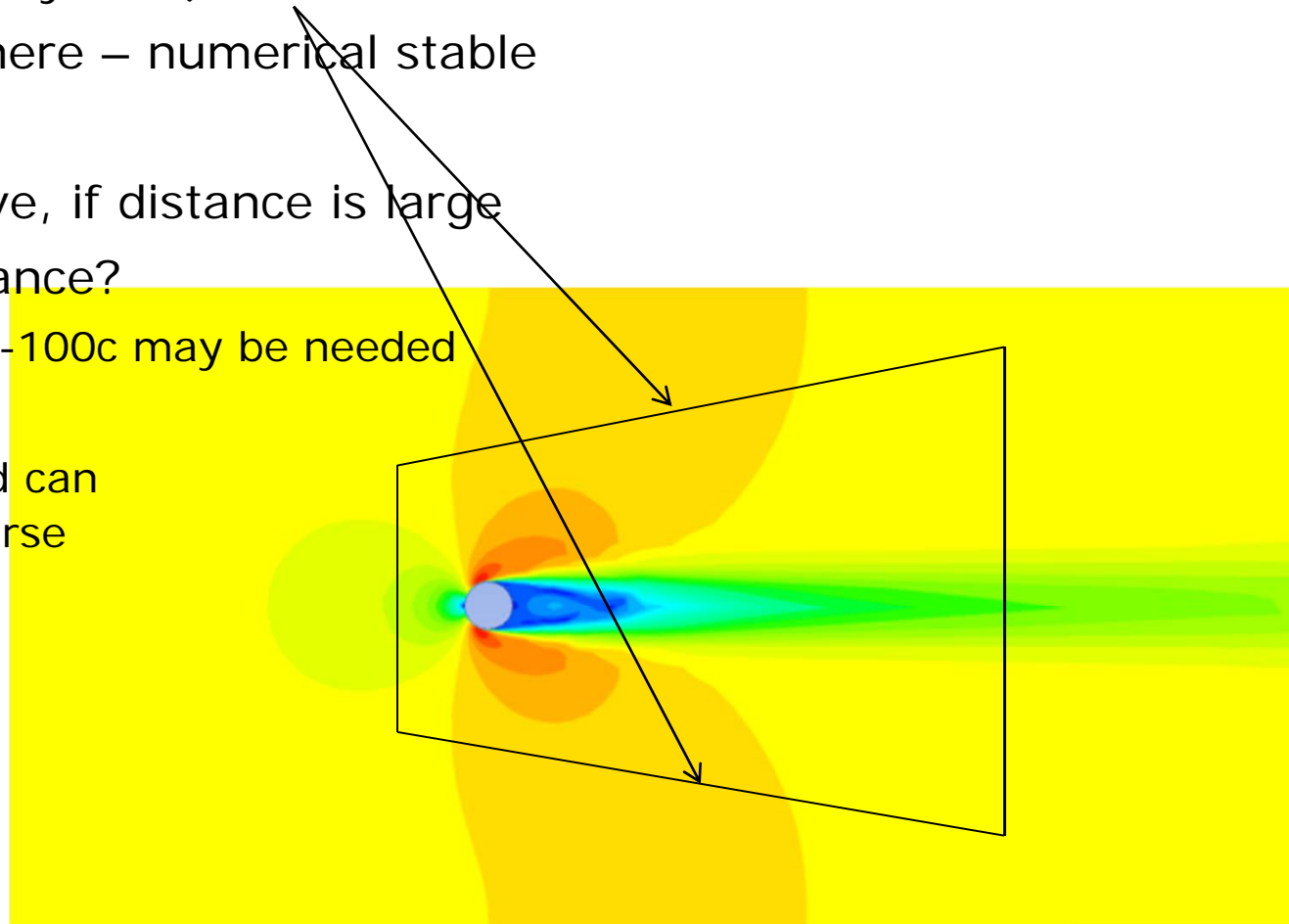
- Velocity and pressure not constant at the boundary
- Might be a local outflow – numerical instability?
- OK if distance is large



External flow B.C.

As inlet BCs (velocity inlet)

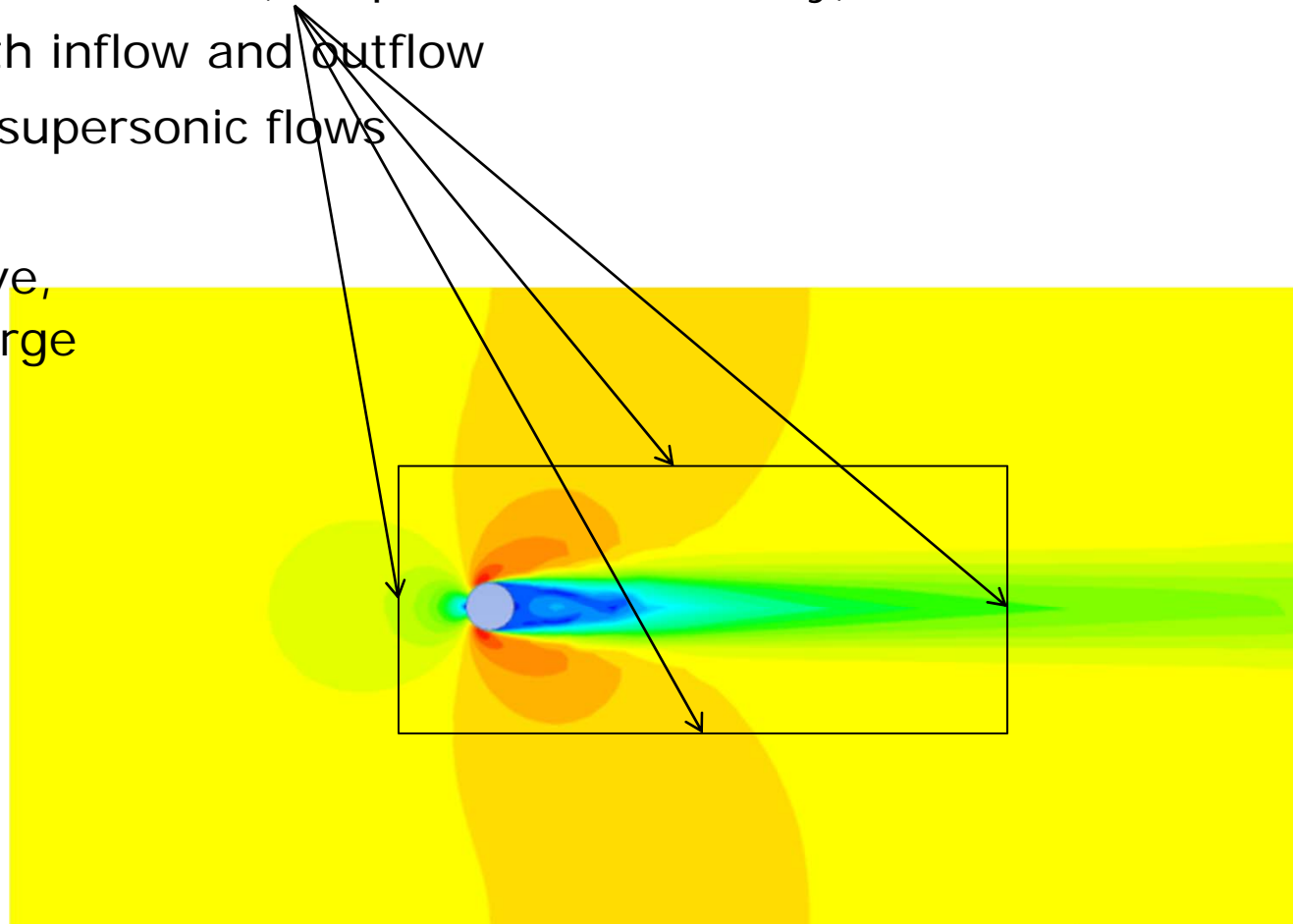
- Inflow everywhere – numerical stable
- Good alternative, if distance is large
- How large distance?
 - 2D wing: 20-100c may be needed
 - 3D: 10-20c
 - Far-field grid can be VERY coarse



External flow B.C.

Far-field boundary condition (compressible flows only)

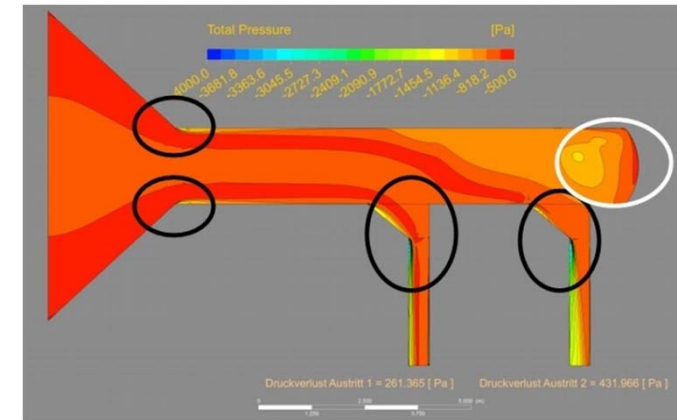
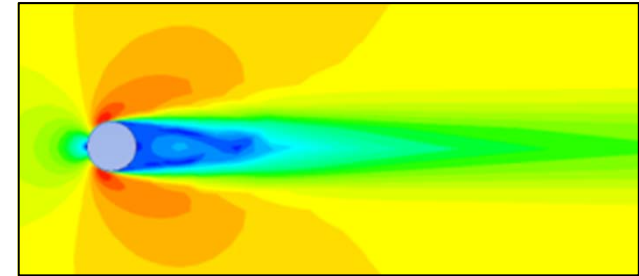
- Can handle both inflow and outflow
- Both sub- and supersonic flows
- Good alternative, if distance is large





Inflow B.C.

- Velocity inlet
 - External and internal flows
- Pressure inlet
 - Sets total (stagnation) condition at inflow
 - Internal flows
- Mass flow inlet
 - Sets mass flow rate over the inflow
 - Internal flows



Inflow conditions for turbulence



- At inflow boundaries the turbulence quantities must be prescribed for turbulent computations
- Inflow turbulence levels mostly not fully known
- Solution may be strongly dependent on the inflow turbulence levels, but mostly only minor dependencies
- Important to prescribe realistic values for solution accuracy and numerical stability

- If problems: move the inflow boundary sufficiently far from the region of interest.
- The turbulence levels at the inflow boundary are also applicable as initial conditions.

Turbulence level

- Turbulence level, Tu

- Relation between rms of fluctuations and mean velocity

$$u_{rms} = Tu U_{\infty}$$

- Assuming isotropic turbulence, turbulence kinetic energy becomes

$$K = \frac{3}{2} (Tu U_{\infty})^2$$

- Must be specified on inlet boundaries and for initial values

- Estimate Tu

- $Tu < 0.3\%$ in external aerodynamic flows
- $Tu \approx 1\%$ in wind tunnels
- $Tu \approx 5 - 10\%$ in internal turbo machinery flows
- $Tu < 2 - 3\%$ usually do not influence the mean flow field



Turbulence length scale

- Viscosity ratio $VR = \nu_T/\nu$
 - Often recommended to be 1 – 10 in external flows
 - Results in length scale

$$L_T = \frac{K^{3/2}}{\varepsilon} = \frac{VR \nu}{C_\mu \sqrt{K}}, \quad \nu_T = C_\mu \frac{K^2}{\varepsilon}, \quad C_\mu = 0.09$$

- Problem: L_T should not be Re dependent
 - For $K - \omega$ models: $\omega \equiv \varepsilon/C_\mu K$
- Turbulence length scale, L_T

- Easier estimated than viscosity ratio
 - $L_T = 1 - 10\%$ of geometrical scales

- Turbulence advection length scale, L_A

$$L_A = \frac{KU_\infty}{\varepsilon} \approx \frac{L_T}{Tu}$$

- The length scale on which free-stream turbulence reduces
 - Should not be much smaller than typical geometrical scales



Transition to turbulence



- The location of the transition point (or region)
 - Depends on surface roughness, free stream turbulence levels, noise, etc.
 - No general method to predict
 - Difficult to measure
- The flow may be dependent on the transition location
 - Try to get information from experiments
 - Try to estimate (specific empirical relations exist)
 - Compute the growth rate of disturbances (a subject as big as CFD)
 - Assume the flow fully turbulent (if transition is unimportant)
- Transition location prescribed in CFD by setting laminar or turbulent walls
 - Laminar: $Re < 10^3$
 - Turbulent: $Re > 10^6$