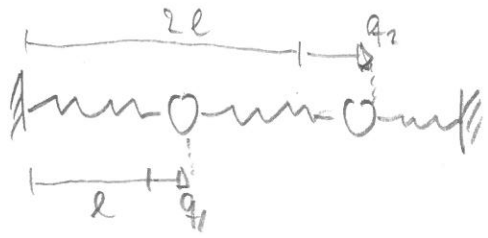


SG 2150

2018-12-20

Lösningförslag

11



$$T = m \frac{\dot{q}_1^2}{2} + m \frac{\dot{q}_2^2}{2}$$

$$V = \frac{k}{2} q_1^2 + \frac{ak}{2} (q_2 - q_1)^2 + \frac{k}{2} (-q_2)^2$$

Eq point: $\frac{\partial V}{\partial q_1} = k[(1+a)q_1^* - aq_2^*] = 0$ $\Rightarrow q_1^* = q_2^* = 0$
 $\frac{\partial V}{\partial q_2} = k[-aq_1^* + (1+a)q_2^*] = 0$

Mass matrix $M = m \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$

Stiffness matrix $K = k \begin{bmatrix} 1+a & -a \\ -a & 1+a \end{bmatrix}$

Eigenvalue problem $(K - \lambda M)\varphi = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$

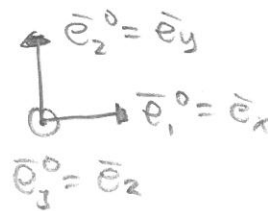
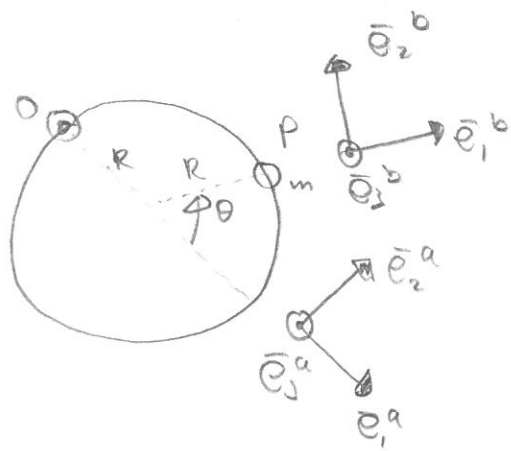
Symmetry $q_1 \rightarrow -q_2$ gives modes are symmetric: $\begin{bmatrix} 1 \\ -1 \end{bmatrix}$
 $q_2 \rightarrow -q_1$ or anti-symmetric: $\begin{bmatrix} 1 \\ 1 \end{bmatrix}$

$\varphi_1 = \begin{bmatrix} 1 \\ 1 \end{bmatrix} \Rightarrow \lambda_1 = \omega_1^2 = \frac{k}{m}$

$\varphi_2 = \begin{bmatrix} 1 \\ -1 \end{bmatrix} \Rightarrow \lambda_2 = \omega_2^2 = (1+2a) \frac{k}{m}$

If $\omega_2 = 2\omega_1$ then $1+2a = 4 \Rightarrow a = \frac{3}{2}$

2



$${}^0\bar{\omega}^a = \omega \bar{e}_z \quad {}^a\bar{\omega}^b = \dot{\theta} \bar{e}_z \quad {}^0\bar{\omega}^b = {}^0\bar{\omega}^a + {}^a\bar{\omega}^b$$

$$\bar{r}_{OP} = R(\bar{e}_1^a + \bar{e}_1^b) \quad \bar{v}_P = \frac{d}{dt} \bar{r}_{OP} = R \left[\frac{d}{dt} \bar{e}_1^a + \frac{d}{dt} \bar{e}_1^b \right] =$$

$$= R \left[\frac{d}{dt} \bar{e}_1^a + {}^0\bar{\omega}^a \times \bar{e}_1^a + \frac{d}{dt} \bar{e}_1^b + {}^0\bar{\omega}^b \times \bar{e}_1^b \right] =$$

$$= R \left[\bar{0} \times \omega \bar{e}_3^a \times \bar{e}_1^a + \bar{0} + (\omega + \dot{\theta}) \bar{e}_3^b \times \bar{e}_1^b \right] = R \left[\omega \bar{e}_2^a + (\omega + \dot{\theta}) \bar{e}_2^b \right]$$

$$T = \frac{m}{2} \bar{v}_P \cdot \bar{v}_P = \frac{mR^2}{2} \left[\omega^2 + (\omega + \dot{\theta})^2 + 2\omega(\omega + \dot{\theta}) \bar{e}_2^a \cdot \bar{e}_2^b \right] =$$

$$= \frac{mR^2}{2} \left[\omega^2 + (\omega + \dot{\theta})^2 + 2\omega(\omega + \dot{\theta}) \cos\theta \right]$$

$\bar{V} = 0 \quad L = T - \bar{V} = T$ No non-conservative forces.

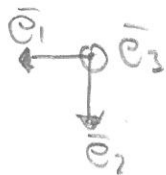
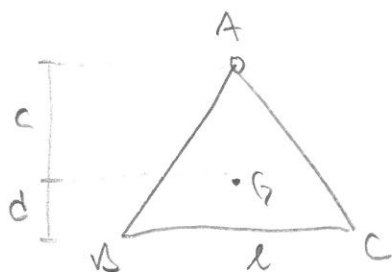
$$p_\theta = \frac{\partial L}{\partial \dot{\theta}} = mR^2 \left[(\omega + \dot{\theta}) + \omega \cos\theta \right] \quad \dot{p}_\theta = mR^2 \left[\ddot{\theta} - \omega \sin\theta \dot{\theta} \right]$$

$$\frac{\partial L}{\partial \theta} = -mR^2 \left[\omega(\omega + \dot{\theta}) \sin\theta \right] \quad \dot{p}_\theta - \frac{\partial L}{\partial \theta} = mR^2 \left[\ddot{\theta} + \omega^2 \sin\theta \right] = 0$$

Differential equation is $\ddot{\theta} + \omega^2 \sin\theta = 0$. Same D.E. as for planar pendulum with $l = \frac{g}{\omega^2}$.

Constant $\theta(t) = \theta_0$ gives $0 + \omega^2 \sin\theta_0 = 0 \Rightarrow \theta_0 = 0$ ($\theta_0 = \pi$ collides with axis)

3



$$d = \frac{1}{3} \frac{\sqrt{3}}{2} l = \frac{1}{2\sqrt{3}} l$$

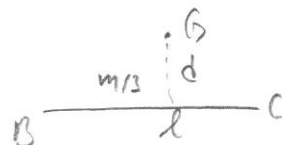
$$c = 2d = \frac{1}{\sqrt{3}} l$$

Planes through the point A with normals \bar{e}_1 or \bar{e}_3 are planes of mirror symmetry. Thus 1, 2, 3 are principal axes for the point A.

First use the point G. Three-fold rotation symmetry about the \bar{e}_3 axis and planar mass distribution gives

$$J_{G1} = J_{G2} = \frac{1}{2} J_{G3}$$

Consider the rod BC.



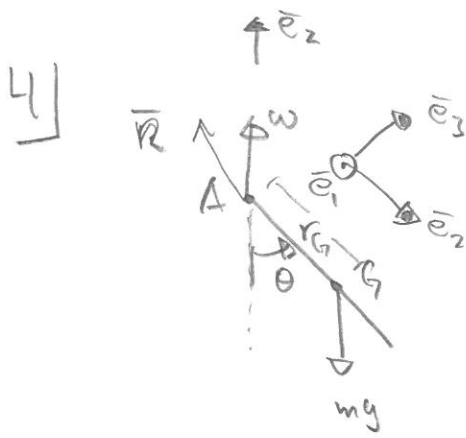
$$J_{G3}^{(BC)} = \frac{m}{3} \frac{l^2}{12} + \frac{m}{3} d^2 = \frac{m}{3} \left(\frac{l^2}{12} + \frac{d^2}{12} \right) = \frac{m}{3} \frac{l^2}{6}$$

$$J_{G3} = 3 \cdot J_{G3}^{(BC)} = \frac{1}{6} m l^2 \Rightarrow J_{G1} = J_{G2} = \frac{1}{12} m l^2$$

$$J_{A1} = J_{G1} + m c^2 = \frac{1}{12} m l^2 + \frac{1}{3} m l^2 = \frac{5}{12} m l^2$$

$$J_{A2} = J_{G2} + m 0^2 = \frac{1}{12} m l^2$$

$$J_{A3} = J_{G3} + m c^2 = \frac{1}{6} m l^2 + \frac{1}{3} m l^2 = \frac{1}{2} m l^2$$



For the given motion,
 \bar{e}_1 stays horizontal and
 θ is constant.

1,2,3 are principal axes.

$$\bar{\omega} = \omega \bar{e}_3 = \omega (-c_\theta \bar{e}_2 + s_\theta \bar{e}_3)$$

$$\bar{L}_A = -J_2 \omega c_\theta \bar{e}_2 + J_3 \omega s_\theta \bar{e}_3$$

$$\begin{aligned} \dot{\bar{L}}_A &= \frac{d}{dt} \bar{L}_A + \bar{\omega} \times \bar{L}_A = \bar{0} + \omega (-c_\theta \bar{e}_2 + s_\theta \bar{e}_3) \times \omega (-J_2 c_\theta \bar{e}_2 + J_3 s_\theta \bar{e}_3) = \\ &= -(J_3 - J_2) \omega^2 s_\theta c_\theta \bar{e}_1. \quad \text{Planar mass} \Rightarrow J_3 - J_2 = J_1 \end{aligned}$$

$$\bar{M}_A = \bar{0} \times \bar{R} + r_G \bar{e}_2 \times (-mg \bar{e}_2) = -mg r_G s_\theta \bar{e}_1$$

$$A \text{ is fixed point: } \dot{\bar{L}}_A = \bar{M}_A \Rightarrow -J_1 \omega^2 s_\theta c_\theta \bar{e}_1 = -mg r_G s_\theta \bar{e}_1$$

$$\text{This is fulfilled if } J_1 \omega^2 s_\theta c_\theta = mg r_G s_\theta. \quad (1)$$

$$\text{For the triangular frame: } J_1 = \frac{5}{12} m l^2 \quad r_G = \frac{1}{\sqrt{3}} l$$

$$\text{For } \theta = \pi/6: s_\theta \neq 0 \quad c_\theta = \frac{\sqrt{3}}{2}$$

$$(1) \Rightarrow \frac{5}{12} m l^2 \omega^2 s_\theta \frac{\sqrt{3}}{2} = mg \frac{1}{\sqrt{3}} l s_\theta \Rightarrow$$

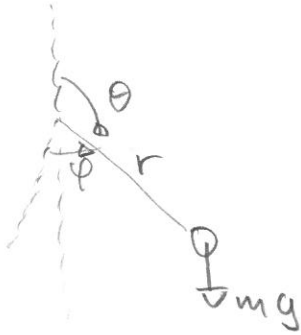
$$\omega^2 = \frac{8}{5} \frac{g}{l}$$

5] $\frac{\partial L}{\partial \varphi} = 0$, φ is cyclic $\Rightarrow \dot{p}_\varphi = 0 \Rightarrow p_\varphi = \frac{\partial L}{\partial \dot{\varphi}} = mr^2 s_\theta^2 \dot{\varphi}$ is constant.

$\frac{\partial L}{\partial t} = 0$; L is time independent $\Rightarrow p_\theta \dot{\theta} + p_\varphi \dot{\varphi} - L =$

$= \frac{m}{2} r^2 [\dot{\theta}^2 + s_\theta^2 \dot{\varphi}^2] + mgr \cos \theta$ is constant.

The Lagrange function is that of a spherical pendulum.



$$6] \quad \text{Let } \mathcal{F} = \int_{t_0}^{t_1} \left[\left(\sum_a p_a \dot{q}_a \right) - H(q, p, t) \right] dt$$

First order variation:

$$\delta \mathcal{F} = \sum_a \int_{t_0}^{t_1} \left[\left(\dot{q}_a - \frac{\partial H}{\partial p_a} \right) \delta p_a + p_a \delta \dot{q}_a - \frac{\partial H}{\partial q_a} \delta q_a \right] dt$$

$$\text{Partial integration} \Rightarrow \int_{t_0}^{t_1} p_a \delta \dot{q}_a dt = \left[p_a \delta q_a \right]_{t_0}^{t_1} - \int_{t_0}^{t_1} \dot{p}_a \delta q_a dt$$

Fixed endpoints $\Rightarrow \delta q_a(t_0) = \delta q_a(t_1) = 0$

$$\delta \mathcal{F} = \sum_a \int_{t_0}^{t_1} \left[\left(\dot{q}_a - \frac{\partial H}{\partial p_a} \right) \delta p_a + \left(-\dot{p}_a - \frac{\partial H}{\partial q_a} \right) \delta q_a \right] dt$$

$\delta \mathcal{F} = 0$ for arbitrary $\delta q_a, \delta p_a \Rightarrow$

$$\dot{q}_a - \frac{\partial H}{\partial p_a} = 0 \quad -\dot{p}_a - \frac{\partial H}{\partial q_a} = 0 \quad \text{for all } t_0 \leq t \leq t_1$$