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Large Eddy Simulation

Basic equations



- Navier-Stokes equations (u, p denote instantaneous values)

$$\frac{\partial u_i}{\partial t} + u_k \frac{\partial u_i}{\partial x_k} = -\frac{1}{\rho} \frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_k} \left(\nu \frac{\partial u_i}{\partial x_k} \right) \quad (1)$$

- RANS - ensemble or time averaging: $u_i(\mathbf{x}, t) = U_i(\mathbf{x}) + u'_i(\mathbf{x}, t)$

$$U_i(\mathbf{x}) = \frac{1}{T} \int_0^T u_i(\mathbf{x}, t') dt' \quad (2)$$

- LES - spatial filtering: $u_i(\mathbf{x}, t) = \bar{u}_i(\mathbf{x}, t) + u'_i(\mathbf{x}, t)$ (in the following, over-bar denotes a filtered quantity)

$$\bar{u}_i(\mathbf{x}, t) = \int_V u_i(\mathbf{x}', t) G(\mathbf{x}, \mathbf{x}') d\mathbf{x}' \quad (3)$$

Basic equations ...

- Filtered Navier-Stokes equations

$$\frac{\partial \bar{u}_i}{\partial t} + \bar{u}_j \frac{\partial \bar{u}_i}{\partial x_j} = -\frac{1}{\rho} \frac{\partial \bar{p}}{\partial x_i} - \frac{\partial \tau_{ij}^{\text{SGS}}}{\partial x_j} + \frac{\partial}{\partial x_j} \left(\nu \frac{\partial \bar{u}_i}{\partial x_j} \right) \quad (4)$$

- Subgrid-scale (SGS) stress tensor

$$\tau_{ij}^{\text{SGS}} = \overline{u_i u_j} - \bar{u}_i \bar{u}_j = L_{ij} + C_{ij} + R_{ij} \quad (5)$$

where the decomposition according to Leonard (1974) is

- $L_{ij} = \overline{\bar{u}_i \bar{u}_j} - \bar{u}_i \bar{u}_j$ - Leonard stresses (interaction between resolved scales)
- $C_{ij} = \overline{\bar{u}_i u'_j} + \overline{u'_i \bar{u}_j}$ - cross terms (interaction between resolved and unresolved scales)
- $R_{ij} = \overline{u'_i u'_j}$ - SGS Reynolds stress (interaction between unresolved small scales)



LES v.s. RANS equations



Filtered N-S (LES)	RANS
Most of the turbulent energy resolved by \bar{u}_i	no turbulence resolved in U_i
u'_i represents only the smallest SGS scales	u'_i represents all turbulent fluctuations
Solution of \bar{u}_i always 3D and time dependent	solution of U_i steady (or slowly varying) and may be homogeneous (or slowly varying) in space

Modelling of SGS turbulence



- SGS represents only small part of turbulence energy
 - Modelling not that critical
 - “resolving at least 80% of turbulence kinetic energy”
- Small scale turbulence more isotropic and in equilibrium
 - Modelling more simple
- Eddy viscosity models are OK
- Filter usually related to grid
 - This means that models are grid dependent by nature (!)
 - Inhomogeneous grid size → inhomogeneous filter
 - Commutation errors (?)

Smagorinsky (1963)



- Eddy viscosity model
- SGS eddy viscosity and length scale relations
- Assumption: SGS production = dissipation
- Subgrid viscosity: $\nu_T^{\text{SGS}} = (C_s \Delta)^2 S$
- One model constant – Smagorinsky constant
 - Standard value 0.18
 - Not very universal

Other models



- One or two-equation models
Solve for SGS turbulence energy
- Dynamic models
Evaluate C_s from coarser filtering – no model constant
Germano (1991)
- Implicit LES
 $\nu_{SGS} \sim \Delta^2$ – as in a 2'nd order numerical scheme
Let the numerical scheme dissipate the SGS energy
MILES – Boris (1992)

Applications for LES and RANS



	LES	RANS
Suitable cases:	free shear flows, blunt bodies, geometry induced separation, low Re number	thin shear layers, weak geometry induced separation, high Re number
Problem cases	thin shear layers (expensive)	massive separation

- But, how to compute a case with both thin boundary layers and massive separation, e.g. a not-so-streamlined car?

Hybrid LES – RANS models

LES and RANS in different regions:

- RANS where LES is too expensive
 - Boundary layers
 - Thin shear layers
- LES where RANS fails
 - Massive separation
- Very active research field
- Many names:
 - DES, PANS, VLES, DDES, IDDES, URANS, XLES, T-RANS, ...



Spalart DES

Spalart et al. 1997

"Detached Eddy Simulation"



- Using the fact that the filtered N-S equation and the RANS equation can be written in the same form - only the modelling of $\overline{u'_i u'_j}$ differs.
- Use a RANS model for $\overline{u'_i u'_j}$ in thin shear layers.
- Use a LES SGS model for $\overline{u'_i u'_j}$ away from walls.

Spalart DES ...

- Starting point is the Spalart-Allmaras 1-eq. model (see notes from RANS modelling).

$$\frac{Dv_T}{Dt} = c_{b1}Sv_T - c_{w1}f_w\left(\frac{v_T}{d}\right)^2 + \mathcal{D}_{v_T}$$

- d is the distance from the wall. If d is replaced with Δ , the filter width, and the balance between production and dissipation is fulfilled (equilibrium hypothesis)

$$c_{b1}Sv_T \approx c_{w1}f_w\left(\frac{v_T}{\Delta}\right)^2$$

or rewritten

$$v_T \approx \frac{c_{b1}}{c_{w1}f_w}\Delta^2S$$

which is similar to the Smagorinsky SGS model



Spalart DES ...

- With $d = \Delta$ the Spalart-Allmaras will work as a SGS model
- Spalart DES

$$d = \min(d, C_{DES}\Delta)$$

where Δ is the local grid size.

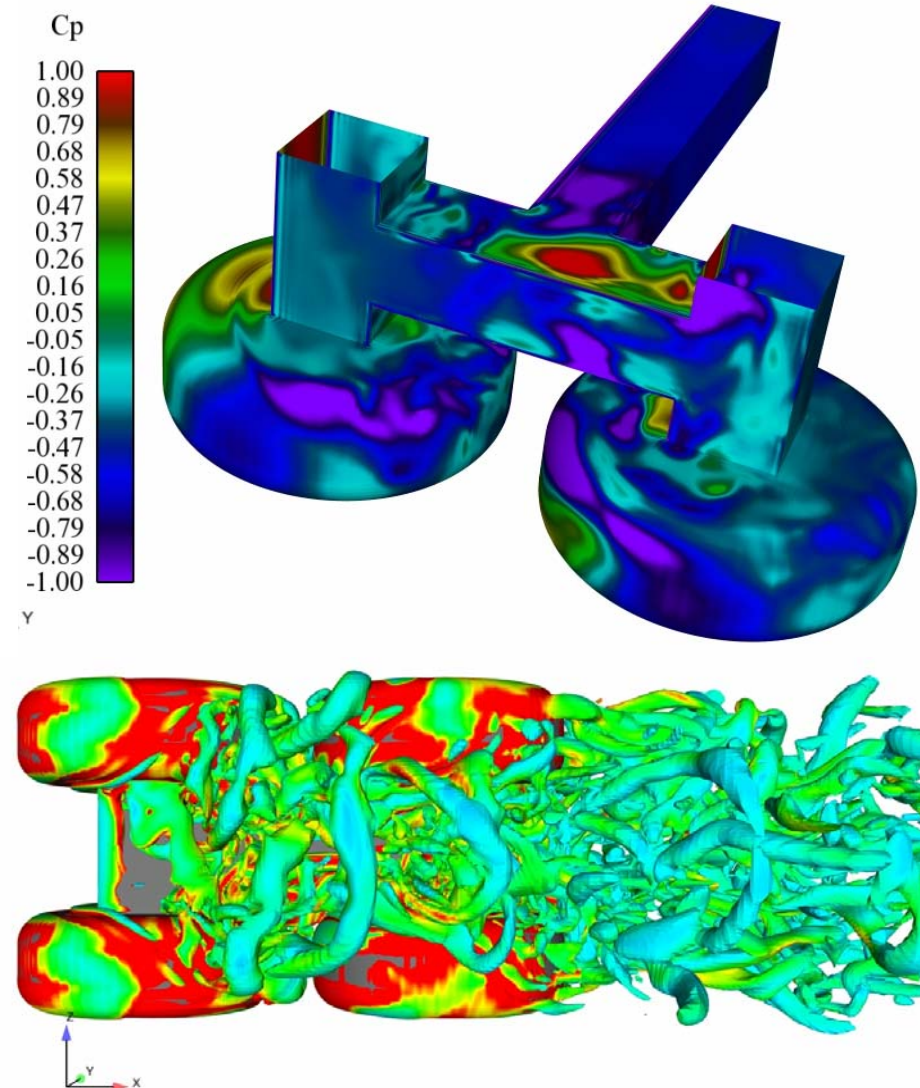
- RANS close to the wall where grid size (in any direction) $>$ wall distance
- LES far from the walls where the grid size $<$ wall distance
- Interface between RANS and LES dependent on the grid
- DES works well in cases with both regions with thin BLs (high Re) and massive separation



Rudimentary landing gear



- Resolved surface pressure (top) and
- Resolved turbulent structures (bottom)
- Hybrid RANS – LES model
 - RANS: here a simple mixing length model



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More examples
Hybrid RANS-LES

